

M/C DESIGN

- Length of open belt drive (O.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 - D_1)^2}{4C}$$

- Length of cross belt driven (C.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 + D_1)^2}{4C}$$

- Velocity ratio

$$V.R = \frac{N_2}{N_1} = \left(\frac{D_1 + t}{D_2 + t} \right) \left(1 - \frac{S}{100} \right)$$

- Ratio of belt tension $\frac{T_1}{T_2} = e^{\mu \theta}$

- Maximum tensile strength of belt

$$T_{\max} = b.t.\sigma_{\text{per}}$$

- Power transmission capacity of belt drive

$$P.T.C = (T_1 - T_2)v$$

- Centrifugal tension $T_c = \frac{T_{\max}}{3}$

- Initial tension $T_0 = \frac{T_1 + T_2 + 2T_c}{2}$

- Number of 'V' belts $n = \frac{P_{\text{total}}}{P_{\text{each}}} \times K_q$

- Buckingham dynamic load

$$F_d = F_t + \frac{20.67v[bc + F_t]}{20.67v + \sqrt{bc + F_t}} \quad c = \frac{a}{K \left(\frac{1}{E_1} + \frac{1}{E_2} \right)}$$

- Soderberg equation $\frac{1}{FOS} = \frac{\sigma_m}{\sigma_y} + \frac{K_f \sigma_v}{\sigma_s}$

- Goodman equation $\frac{1}{FOS} = \frac{K_f \sigma_m}{\sigma_{ut}} + \frac{K_f \sigma_v}{\sigma_e}$

- Notch sensitivity index $q = \frac{K_f - 1}{K_t - 1}$

- Unwin's formula $d = 6\sqrt{t}$

- Area of transverse fillet weld

$$A_{T.F.W} = h\ell_s \quad h = \frac{t}{\cos \theta + \sin \theta}$$

- Strength of T.F.W $= 0.832t\ell_s\tau_{\text{per}}$

- Strength of parallel fillet weld $= 0.707t\ell_s\tau_{\text{per}}$

- Strength of Butt weld $= h\ell(\sigma_t)_{\text{per}}$

- Frictional torque of thrust bearing by uniform pressure theory

$$(T_f)_{\text{UPT}} = \frac{2}{3}\mu\omega \left(\frac{R_o^3 - R_i^3}{R_o^2 - R_i^2} \right)$$

- Frictional torque of thrust bearing by uniform wear theory

$$(T_f)_{\text{UWT}} = \mu\omega \left(\frac{R_o + R_i}{2} \right)$$

- Power loss $P_{\text{loss}} = T_f.\omega$

- T_f equations for flat pivot bearing

$$(T_f)_{\text{UPT}} = \frac{2}{3}\mu WR$$

$$(T_f)_{\text{UWT}} = \frac{1}{2}\mu WR$$

- Bearing pressure $p_{\text{ind}} = \frac{W}{LD}$

- Strength of bearing $= p_{\text{per}} \times L \times D$

- Bearing characteristic number $= \frac{zn}{p}$

- Mc-Kee's equation

$$\mu = \frac{33}{10^8} \left[\left(\frac{zn'}{p'} \right) \left(\frac{D}{C} \right) \right] + K$$

- Somer field No. $S = \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)^2$

- Petroff's equation for $\mu = \frac{2\pi^2}{p} \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)$

- Life of bearing $L_{50} = \left(\frac{C}{P_e} \right)^k$ (Million rev.)

K_q = service or overload factor.

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