

M/C DESIGN

- Length of open belt drive (O.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 - D_1)^2}{4C}$$

- Length of cross belt driven (C.B.D)

$$= 2C + \frac{\pi}{2}(D_1 + D_2) + \frac{(D_2 + D_1)^2}{4C}$$

- Velocity ratio

$$V.R = \frac{N_2}{N_1} = \left(\frac{D_1 + t}{D_2 + t} \right) \left(1 - \frac{S}{100} \right)$$

- Ratio of belt tension

$$\frac{T_1}{T_2} = e^{\mu \theta}$$

- Maximum tensile strength of belt

$$T_{max} = b.t.\sigma_{per}$$

- Power transmission capacity of belt drive

$$P.T.C = (T_1 - T_2)v$$

- Centrifugal tension

$$T_c = \frac{T_{max}}{3}$$

- Initial tension

$$T_0 = \frac{T_1 + T_2 + 2T_c}{2}$$

- Number of 'V' belts

$$n = \frac{P_{total}}{P_{each}} \times K_q$$

- Buckingham dynamic load

$$F_d = F_1 + \frac{20.67v[bc + F_1]}{20.67v + \sqrt{bc + F_1}}$$

$$c = \frac{a}{K \left(\frac{1}{E_1} + \frac{1}{E_2} \right)}$$

- Soderberg equation

$$\frac{1}{FOS} = \frac{\sigma_m}{\sigma_y} + \frac{K_f \sigma_v}{\sigma_s}$$

- Goodman equation

$$\frac{1}{FOS} = \frac{K_1 \sigma_m}{\sigma_{ut}} + \frac{K_f \sigma_v}{\sigma_e}$$

- Notch sensitivity index

$$q = \frac{K_f - 1}{K_t - 1}$$

- Unwin's formula

$$d = 6\sqrt{t}$$

- Area of transverse fillet weld

$$A_{TFW} = h\ell_w, \quad h = \frac{t}{\cos \theta + \sin \theta}$$

- Strength of T.F.W = $0.832t\ell_w\tau_{per}$

- Strength of parallel fillet weld

$$= 0.707t\ell_w\tau_{per}$$

- Strength of Butt weld = $h\ell(\sigma_t)_{per}$

- Frictional torque of thrust bearing by uniform pressure theory

$$(T_f)_{UPT} = \frac{2}{3}\mu\omega \left(\frac{R_o^3 - R_i^3}{R_o^2 - R_i^2} \right)$$

- Frictional torque of thrust bearing by uniform wear theory

$$(T_f)_{UWT} = \mu\omega \left(\frac{R_o + R_i}{2} \right)$$

- Power loss $P_{loss} = T_f.\omega$

- T_f equations for flat pivot bearing

$$(T_f)_{UPT} = \frac{2}{3}\mu WR$$

$$(T_f)_{UWT} = \frac{1}{2}\mu WR$$

- Bearing pressure

$$p_{ind} = \frac{W}{LD}$$

- Strength of bearing = $p_{per} \times L \times D$

- Bearing characteristic number = $\frac{zn}{p}$

- Mc-Kee's equation

$$\mu = \frac{33}{10^8} \left[\left(\frac{zn'}{p'} \right) \left(\frac{D}{C} \right) \right] + K$$

- Somer field No.

$$S = \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)^2$$

- Petroff's equation for μ

$$\mu = \frac{2\pi^2}{p} \left(\frac{zn}{p} \right) \left(\frac{D}{C} \right)$$

- Life of bearing

$$L_{50} = \left(\frac{C}{P_e} \right)^k \quad (\text{Million rev.})$$

K_q = service or overload factor.

Mechanical Engineering Formulas Cheat Sheet

Kurt Gieck, Reiner Gieck



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The Academic Discourse of Mechanical Engineering Thi Ngoc Phuong Le, Minh Man Pham, Michael Barlow, 2023-03-15 This volume examines rhetorical conventions employed in mechanical engineering research to understand the knowledge making principles of the discipline as well as their expression within the research article In particular the study analyses the organisational patterns of mechanical engineering research articles using Swales's conceptualisation of moves and steps In addition the research identifies the phraseology associated with specific moves and steps The study draws on a corpus of 120 mechanical engineering research articles equally distributed across two sub disciplines mechanical systems and thermal fluids engineering three research traditions experimental theoretical and mixed methods and two publication periods 2002-2006 and 2012-2016 It adopts an integrated methodology intertwining various approaches and perspectives including corpus linguistics move analysis discourse analysis and interviews to address two main strands of research enquiry i What are the properties of the rhetorical structures in terms of range frequency and length for each section of mechanical engineering research articles ii What effect does sub discipline research tradition and publication date have on the rhetorical structure of research articles

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